

We claim:

1           1.     A method of setting the drive and sense frequencies of a gyroscope  
 2     having a drive mass and a sense mass coupled together by a flexure assembly  
 3     comprising:  
 4           selecting a drive stiffness,  $K_d$ ;  
 5           selecting geometric parameters of said flexure assembly to obtain a desired drive  
 6     frequency,  $\omega_d$ ;  
 7           selecting a configurational parameter of said flexure assembly to obtain a desired  
 8     sense frequency,  $\omega_s$ ; and  
 9           determining whether said gyroscope has obtained desired performance and size  
 10    envelope characteristics.

1           2.     The method of claim 1 further comprising repeating selecting a drive  
 2     stiffness,  $K_d$ ; selecting geometric parameters of said flexure assembly to obtain a  
 3     desired drive frequency,  $\omega_d$ ; and selecting a configurational parameter of said flexure  
 4     assembly to obtain a desired sense frequency,  $\omega_s$ ; until it is determined that said  
 5     gyroscope has obtained desired performance and size envelope characteristics.

1           3.     The method of claim 1 where selecting geometric parameters of said  
2 flexure assembly to obtain a desired drive frequency,  $\omega_d$ , comprises selecting length  
3 and/or width of at least one individual flexure within said flexure assembly.

1           4.     The method of claim 3 where selecting length and width of at least one  
2 individual flexure within said flexure assembly comprises selecting length and/or width  
3 of each individual flexure within said flexure assembly.

1           5.     The method of claim 1 where selecting a configurational parameter of said  
2 flexure assembly to obtain a desired sense frequency,  $\omega_s$ , comprises selecting an  
3 orientation of at least one flexure within said flexure assembly relative to other ones of  
4 said flexures with said flexure assembly.

1           6.     The method of claim 5 where flexures within said flexure assembly are  
2 oriented symmetrically about an axis of symmetry of said gyroscope, and where  
3 selecting an orientation of at least one flexure within said flexure assembly relative to  
4 other ones of said flexures with said flexure assembly comprises selecting one of a  
5 possible number of orientations of said at least one flexure to said axis of symmetry of  
6 said gyroscope.

1           7.     The method of claim 1 where said flexure assembly includes at least one  
2 pair of flexures, and where selecting a configurational parameter of said flexure

3 assembly to obtain a desired sense frequency,  $\omega_s$ , comprises selecting an angle which  
4 said pair of flexures makes to each other.

1 8. The method of claim 7 where said flexure assembly comprises two  
2 diametrically opposing pairs of flexures and where selecting an angle which said pair of  
3 flexures makes to each other comprises setting a dihedral angle between each of said  
4 flexures of said two diametrically opposing pairs.

1 9. The method of claim 1 where selecting geometric parameters of said  
2 flexure assembly to obtain a desired drive frequency,  $\omega_d$ , comprises selecting length, L,  
3 and width, w, of four flexures formed into two pairs comprising said flexure assembly,  
4 where

$$\omega_d^2 = \frac{4 E w^3 t R^2}{12 L^3 I_d}$$

6 where E is the Young's modulus of said flexure, t is the process thickness of said  
7 flexure,  $I_d$  is the rotational moment of inertia of said drive mass about a rate axis, and R  
8 is the radius of said drive mass, where said drive mass is a ring-shaped mass.

1 10. The method of claim 1 where selecting a configurational parameter of said  
2 flexure assembly to obtain a desired sense frequency,  $\omega_s$ , comprises selecting  $\theta$  in

$$\omega_s^2 = \frac{4 E w t^3 \sin \theta R^2}{12 L^3 I_s}$$

4 where E is the Young's modulus of said flexure, t is the process thickness of said  
 5 flexure,  $I_s$  is the rotational moment of inertia of said sense mass about a sense axis, R is  
 6 the radius of said drive mass, where said drive mass is a ring-shaped mass, L is the  
 7 length of each flexure within said flexure assembly, and w is the width of each flexure  
 8 within said flexure assembly which is comprised of four flexures formed into two pairs.

1 11. The method of claim 9 where selecting a configurational parameter of said  
 2 flexure assembly to obtain a desired sense frequency,  $\omega_s$ , comprises selecting  $\theta$  in

3 
$$\omega_s^2 = \frac{4 E w t^3 \sin \theta R^2}{12 L^3 I_s}.$$

1 12. An improvement in a gyroscope comprising:

2 a drive mass;

3 a sense mass; and

4 a flexure assembly coupling said drive and sense mass together;

5 where said drive mass has a selecting drive stiffness,  $K_d$  obtained by selecting  
 6 geometric parameters of said flexure assembly to obtain a desired drive frequency,  $\omega_d$ ;  
 7 and where said sense mass as a sense stiffness  $K_s$  obtained by selecting a  
 8 configurational parameter of said flexure assembly to obtain a desired sense frequency,  
 9  $\omega_s$ , and where said gyroscope has obtained desired performance and size envelope  
 10 characteristics by independent selection of said geometric and configurational  
 11 parameters of said flexure assembly.

1           13.    The improvement of claim 12 where said geometric parameters of said  
2   flexure assembly selected to obtain a desired drive frequency,  $\omega_d$ , comprise length  
3   and/or width of at least one individual flexure within said flexure assembly.

1           14.    The improvement of claim 13 where said selected length and width of at  
2   least one individual flexure within said flexure assembly comprises a selected length  
3   and/or width of each individual flexure within said flexure assembly.

1           15.    The improvement of claim 12 where said configurational parameter of said  
2   flexure assembly selected to obtain a desired sense frequency,  $\omega_s$ , comprises a  
3   selected orientation of at least one flexure within said flexure assembly relative to other  
4   ones of said flexures with said flexure assembly.

1           16.    The improvement of claim 15 where flexures within said flexure assembly  
2   are oriented symmetrically about an axis of symmetry of said gyroscope, and where  
3   said a selected orientation of at least one flexure within said flexure assembly relative to  
4   other ones of said flexures with said flexure assembly comprises a selected one of a  
5   possible number of orientations of said at least one flexure to said axis of symmetry of  
6   said gyroscope.

1           17.    The improvement of claim 12 where said flexure assembly includes at  
2   least one pair of flexures, and where said configurational parameter of said flexure

3 assembly selected to obtain a desired sense frequency,  $\omega_s$ , comprises a selected angle  
4 which said pair of flexures makes to each other.

1 18. The improvement of claim 17 where said flexure assembly comprises two  
2 diametrically opposing pairs of flexures and where said angle which said pair of flexures  
3 makes to each other comprises a selected dihedral angle between each of said flexures  
4 of said two diametrically opposing pairs.

1 19. The improvement of claim 12 where said geometric parameters of said  
2 flexure assembly selected to obtain a desired drive frequency,  $\omega_d$ , comprises a length,  
3 L, and width, w, of four flexures formed into two pairs comprising said flexure assembly,  
4 where

$$\omega_d^2 = \frac{4 E w^3 t R^2}{12 L^3 I_d}$$

6 where E is the Young's modulus of said flexure, t is the process thickness of said  
7 flexure,  $I_d$  is the rotational moment of inertia of said drive mass about a rate axis, and R  
8 is the radius of said drive mass, where said drive mass is a ring-shaped mass.

1 20. The improvement of claim 12 where said configurational parameter of said  
2 flexure assembly selected to obtain a desired sense frequency,  $\omega_s$ , comprises a  
3 selected  $\theta$  in

$$\omega_s^2 = \frac{4 E w t^3 \sin \theta R^2}{12 L^3 I_s}$$

5           where E is the Young's modulus of said flexure, t is the process thickness of said  
6           flexure,  $I_s$  is the rotational moment of inertia of said sense mass about a sense axis, R is  
7           the radius of said drive mass, where said drive mass is a ring-shaped mass, L is the  
8           length of each flexure within said flexure assembly, and w is the width of each flexure  
9           within said flexure assembly which is comprised of four flexures formed into two pairs.

1           21.    The improvement of claim 19 where said configurational parameter of said  
2           flexure assembly selected to obtain a desired sense frequency,  $\omega_s$ , comprises a  
3           selected  $\theta$  in

$$\omega_s^2 = \frac{4 E w t^3 \sin \theta R^2}{12 L^3 I_s}.$$